

## Introduction

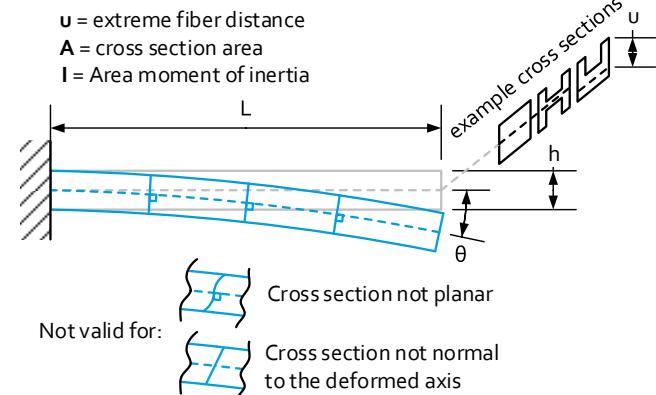
Beam theory is such a common engineering fundamental; it is impossible to be omitted from almost any engineering specialism. However, this sheet incorporates stress and stiffness as well. For more information on, and calculations of the area moment of inertia  $I$ , see sheet: *Area moment of inertia*.

## Validity

Euler-Bernoulli beam theory is only valid with the following assumptions:

- Cross sections of the beam do not deform in a significant manner under the application of transverse or axial loads and can be assumed as rigid.
- During deformation, the cross section of the beam is assumed to remain planar and normal to the deformed axis of the beam.

Generally, these criteria are met when the beam is a slender beam with small rotations. This means the slenderness of the beam (ratio  $L/h$ ) should be larger than 10 and the rotation of the neutral axis ( $\theta$ ) should be smaller than  $5^\circ$ .



Load case	Curvature $\theta$ Sag $\delta y$	Reaction force R Shear force D Reaction moment $M_R$	Stress $\sigma$ Stiffness C or K
1	$\theta_A = 0$ $\theta_B = \frac{FL^2}{2EI_x}$ $\delta y_z = -\frac{(Fz^2)(3L-z)}{6EI_x}$ $\delta y_{max} = -\frac{FL^3}{3EI_x}$ @ z = L	$R_A = F$ $R_B = N.A.$ $D_z = F$ $M_{Rz} = F(z-L)$ $M_{Rmax} = -FL$ @ z = 0	$ \sigma_z  = \frac{ M_{Rz} }{I_x} u = -\frac{ F u(z-L)}{I_x}$ $ \sigma_{max}  = \frac{ F Lu}{I_x}$ @ z = 0 $ C_y  = \left  \frac{F}{\delta y_{z(F)}} \right  = \frac{3EI_x}{L^3}$ @ z = L
2	$\theta_A = 0$ $\theta_B = \frac{ML}{EI_x}$ $\delta y_z = -\frac{Mz^2}{2EI_x}$ $\delta y_{max} = -\frac{ML^2}{2EI_x}$ @ z = L	$R_A = 0$ $R_B = N.A.$ $D_z = N.A.$ $M_{Rz} = M$ $M_{Rmax} = M$ @ z = const.	$ \sigma_z  = \frac{ M_{Rz} }{I_x} u = \frac{ M u}{I_x}$ $ \sigma_{max}  = \frac{ M u}{I_x}$ @ z = const. $ K_y  = \left  \frac{M}{\theta_B} \right  = \frac{EI_x}{L}$ @ z = L
3	$\theta_A = 0$ $\theta_B = \frac{qL^3}{6EI_x}$ $\delta y_z = -\frac{qz^2(6L^2-4Lz+z^2)}{24EI_x}$ $\delta y_{max} = -\frac{qL^4}{8EI_x}$ @ z = L	$R_A = qL$ $R_B = N.A.$ $D_z = q(L-z)$ $M_{Rz} = -\frac{q(L-z)^2}{2}$ $M_{Rmax} = -\frac{qL^2}{2}$ @ z = 0	$ \sigma_z  = \frac{ M_{Rz} }{I_x} u = \frac{ q u(L-z)^2}{2I_x}$ $ \sigma_{max}  = \frac{ q L^2u}{2I_x}$ @ z = 0 $ C_y  = \left  \frac{q}{\delta y_{max}} \right  = \frac{8EI_x}{L^4}$ @ z = L
4	$\theta_A = \frac{Fab(L+b)}{6EI_x L}$ $\theta_B = \frac{Fab(L+a)}{6EI_x L}$ $\delta y_z = -\frac{Fab^2}{6EI_x} \left[ \left(1 + \frac{L}{b}\right) \frac{z}{L} - \frac{z^3}{abL} \right]$ $\delta y_{max} = -\frac{Fab\sqrt{(L^2-b^2)^3}}{9\sqrt{3}EI_x L}$ $@ z = \sqrt{\frac{L^2-b^2}{3}}$ only if a > b	$R_A = \frac{Fb}{L}$ $R_B = \frac{Fa}{L}$ $D_z = -\frac{Fb}{L}$ $M_{Rz} = \frac{Fbz}{a+b}$ $M_{Rmax} = \frac{Fba}{a+b}$ @ z = a	$ \sigma_z  = \frac{ M_{Rz} }{I_x} u = \frac{ F bz u}{I_x L}$ $ \sigma_{max}  = \frac{ F bau}{I_x(a+b)}$ @ z = a $ C_y  = \left  \frac{F}{\delta y_{z(F)}} \right  = \frac{3EI_x L}{a^2 b^2}$ @ z = a
	$\theta_A = \frac{Fab(L+b)}{6EI_x L}$ $\theta_B = \frac{Fab(L+a)}{6EI_x L}$ $\delta y_z = -\frac{Fa^2 b}{6EI_x} \left[ \left(1 + \frac{L}{a}\right) \frac{L-z}{L} - \frac{(L-z)^3}{abL} \right]$ $\delta y_{max} = -\frac{Fa\sqrt{(L^2-a^2)^3}}{9\sqrt{3}EI_x L}$ $@ z = L - \sqrt{\frac{L^2-a^2}{3}}$ only if a < b	$R_A = \frac{Fb}{L}$ $R_B = \frac{Fa}{L}$ $D_z = \frac{Fb}{L} - F$ $M_{Rz} = \left( \frac{Fbz}{L} \right) - F(z-a)$ $M_{Rmax} = \frac{Fba}{L}$ @ z = a	$ \sigma_z  = \frac{ M_{Rz} }{I_x} u = \frac{\left( \frac{Fbz}{L} - F(z-a) \right) u}{I_x}$ $ \sigma_{max}  = \frac{ F bau}{I_x L}$ @ z = a $ C_y  = \left  \frac{F}{\delta y_{z(F)}} \right  = \frac{3EI_x L}{a^2 b^2}$ @ z = a

Load case	Curvature $\theta$ sag $\delta y$	Reaction force R Shear force D Reaction moment $M_R$	Stress $\sigma$ Stiffness C
<p>5</p>	$\theta_A = \frac{ML}{6EI_x}$ $\theta_B = \frac{ML}{3EI_x}$ $\delta y_z = -\frac{ML^2}{6EI_x} \left[ \frac{z}{L} - \left( \frac{z}{L} \right)^3 \right]$ $\delta y_{max} = -\frac{ML^2}{9\sqrt{3}EI_x} @ z = \frac{L}{\sqrt{3}}$	$R_A = \frac{M}{L}$ $R_B = -\frac{M}{L}$ $D_z = \frac{M}{L}$ $M_{Rz} = \left( \frac{M}{L} \right) z$ $M_{Rmax} = M @ z = L$	$ \sigma_z  = \frac{ M_{Rz} }{I_x} u = \frac{ M zu}{L I_x}$ $ \sigma_{max}  = \frac{ M u}{I_x} @ z = L$ $ C_y  = \left  \frac{M}{\theta_B} \right  = \frac{3EI_x}{L} @ z = L$
<p>6</p>	$\theta_A = \frac{qL^3}{24EI_x}$ $\theta_B = \frac{qL^3}{24EI_x}$ $\delta y_z = -\frac{qL^4}{24EI_x} \left[ \frac{z}{L} - 2 \left( \frac{z}{L} \right)^3 + \left( \frac{z}{L} \right)^4 \right]$ $\delta y_{max} = -\frac{5}{384} \frac{qL^4}{EI_x} @ z = \frac{L}{2}$	$R_A = \frac{qL}{2}$ $R_B = \frac{qL}{2}$ $D_z = \frac{qL}{2} - qz$ $M_{Rz} = \frac{qz(L-z)}{2}$ $M_{Rmax} = \frac{qL^2}{8} @ z = \frac{L}{2}$	$ \sigma_z  = \frac{ M_{Rz} }{I_x} u = \frac{ qz(L-z)u }{2I_x}$ $ \sigma_{max}  = \frac{ q L^2u}{8I_x} @ z = \frac{L}{2}$ $ C_y  = \left  \frac{q}{\delta y_{max}} \right  = \frac{384EI_x}{5L^4} @ z = \frac{L}{2}$
<p>7</p>	<p>0 ≤ z ≤ a</p> $\theta_A = \frac{FbL}{6EI_x}$ $\theta_B = \frac{FbL}{3EI_x}$ $\delta y_z = \frac{Fba^2}{6EI_x} \left[ \frac{z}{a} - \left( \frac{z}{a} \right)^3 \right]$ $\delta y_{max} = \frac{Fba^2}{9\sqrt{3}EI_x} @ z = \frac{a}{\sqrt{3}}$ <p>a ≤ z ≤ L</p> $\theta_C = \frac{Fb(2a+3b)}{3EI_x}$ $\delta y_z = -\frac{F((z-a)+a)}{6EI_x} [ab - 3bz + z^2 - 2az + a^2]$ $\delta y_{max} = -\frac{Fb^2L}{3EI_x} @ z = L$	$R_A = -\frac{Fb}{a}$ $R_B = F + \frac{Fb}{a}$ $D_z = -\left( \frac{Fb}{a} \right)$ $M_{Rz} = -\frac{Fbz}{a}$ $M_{Rmax} = -Fb @ z = a$	$ \sigma_z  = \frac{ M_{Rz} }{I_x} u = \frac{ F bzu}{aI_x}$ $ \sigma_{max}  = \frac{ F bu}{I_x} @ z = a$ $ C_y  = \left  \frac{F}{\delta y_{z(F)}} \right  = \frac{3EI_x}{b^2L} @ z = L$
<p>8</p>	<p>0 ≤ z ≤ a</p> $\theta_A = \frac{qb^2a}{12EI_x}$ $\theta_B = \frac{qb^2a}{6EI_x}$ $\delta y_z = \frac{qb^2a^2}{12EI_x} \left[ \frac{z}{a} - \left( \frac{z}{a} \right)^3 \right]$ $\delta y_{max} = \frac{qb^2a^2}{18\sqrt{3}EI_x} @ z = \frac{a}{\sqrt{3}}$ <p>a ≤ z ≤ L</p> $\theta_C = \frac{qb^2L}{6EI_x}$ $\delta y_z = -\frac{qb^4}{24EI_x} \left[ 4 \frac{a}{b} \frac{z-a}{b} + 6 \left( \frac{z-a}{b} \right)^2 - 4 \left( \frac{z-a}{b} \right)^3 + \left( \frac{z-a}{b} \right)^4 \right]$ $\delta y_{max} = -\frac{qb^3(4a+3b)}{24EI_x} @ z = L$	$R_A = -\frac{qb^2}{2a}$ $R_B = \frac{qb(b+2a)}{2a}$ $D_z = -\frac{qb^2}{2a}$ $M_{Rz} = -\left( \frac{qb^2}{2a} \right) z$ $M_{Rmax} = -\frac{qb^2}{2} @ z = a$	$ \sigma_z  = \frac{ M_{Rz} }{I_x} u = \frac{ qb^2zu}{2aI_x}$ $ \sigma_{max}  = \frac{ qb^2u}{2I_x} @ z = a$ $ C_y  = \left  \frac{q}{\delta y_{max}} \right  = \frac{18\sqrt{3}EI_x}{b^2a^2} @ z = \frac{a}{\sqrt{3}}$

Load case	Curvature $\theta$ sag $\delta y$	Reaction force R Shear force D Moment M	Stress $\sigma$ Stiffness C
9	$\theta_A = \frac{Fab^2}{4EI_x L}$ $\theta_B = 0$ $\delta y_z = -\frac{Fb^2}{4EI_x} \left[ \frac{az}{L^2} - \frac{2}{3} \left( 1 + \frac{a}{2L} \right) \left( \frac{z}{L} \right)^3 \right]$ $\delta y_{max} = -\frac{ab^2 F}{6EI_x} \sqrt{\frac{a}{2L+a}} @ z = L \cdot \sqrt{\frac{a}{1+\frac{a}{2L}}}$ Only if $a \geq 0.414L$	$R_A = F \left( \frac{b}{L} \right)^2 \left( 1 + \frac{a}{2L} \right)$ $R_B = F \left( \frac{a}{L} \right)^2 \left( 1 + \frac{b}{2L} + \frac{3}{2} \frac{b}{a} \right)$ $D_z = F \left( \frac{b}{L} \right)^2 \left( 1 + \frac{a}{2L} \right)$ $M_{Rz} = Fz \left( \frac{b}{L} \right)^2 \left( 1 + \frac{a}{2L} \right)$ $M_{Rmax} = \frac{Fab^2}{L^2} \left( 1 + \frac{a}{2L} \right) @ z = a, \text{ with } a \leq 0.414L$	$ \sigma_z  = \frac{ M_{Rz} }{I_x} u = \frac{ F zb^2 u(2L+a)}{2L^3 I_x}$ $ \sigma_{max}  = \frac{ F ab^2 u(1+\frac{a}{2L})}{L^2 I_x} @ z=a$ $ C_y  = \left  \frac{F}{\delta y_{z(F)}} \right  = \left  \frac{12EI_x L^3}{b^2 a^2 [3L^2 - 2aL - a^2]} \right  @ z=a$
10	$\theta_A = \frac{Fab^2}{4EI_x L}$ $\theta_B = 0$ $\delta y_z = \frac{Fa(L-z)^2}{12L^3} \frac{2La^2 - 3L^2 z + a^2 z}{EI_x}$ $\delta y_{max} = \delta y_z(z_{max})$ $@ z_{max} = L \cdot \left[ \frac{2al+bL-ba}{2al+3bL+ba} \right]$ Only if $a \leq 0.414L$	$R_A = F \left( \frac{b}{L} \right)^2 \left( 1 + \frac{a}{2L} \right)$ $R_B = F \left( \frac{a}{L} \right)^2 \left( 1 + \frac{b}{2L} + \frac{3}{2} \frac{b}{a} \right)$ $D_z = -F \left( \frac{a}{L} \right)^2 \left( 1 + \frac{b}{2L} + \frac{3}{2} \frac{b}{a} \right)$ $M_{Rz} = Fz \left( \frac{b}{L} \right)^2 \left( 1 + \frac{a}{2L} \right) - F(z-a)$ $M_{Rmax} = -F \frac{ab}{L} \left( 1 - \frac{b}{2L} \right)$ $@ z = L, \text{ with } a \geq 0.414L$	$ \sigma_z  = \frac{ M_{Rz} }{I_x} u = \frac{ F 2Lb^2 + b^2 za - 2L^3 z + 2L^3 a}{2L^3 I_x}$ $ \sigma_{max}  = \frac{ Fabu(1-\frac{b}{2L}) }{L^2 I_x} @ z=L$ $ C_y  = \left  \frac{F}{\delta y_{z(F)}} \right  = \left  \frac{12EI_x L^3}{b^2 a^2 [3L^2 - 2aL - a^2]} \right  @ z=a$
11	$\theta_A = 0$ $\theta_B = 0$ $\delta y_z = -\frac{FLb^2}{6EI_x} \left[ 3 \frac{a}{L} \left( \frac{z}{L} \right)^2 - \left( 1 + \frac{2a}{L} \right) \left( \frac{z}{L} \right)^3 \right]$ $\delta y_{max} = -\frac{2Fa^2 b^3}{3EI_x L^2} \left( \frac{1}{1+\frac{2b}{L}} \right)^2 @ z = L - L * \frac{1}{1+\frac{2b}{L}}$	$R_A = F \left( \frac{b}{L} \right)^2 \left( 1 + \frac{2a}{L} \right)$ $R_B = F \left( \frac{a}{L} \right)^2 \left( 1 + \frac{2b}{L} \right)$ $D_z = F \left( \frac{b}{L} \right)^2 \left( 1 + \frac{2a}{L} \right)$ $M_{Rz} = \frac{Fb^2 (zL+2za+aL)}{L^3}$ $M_{Rmax} = -\frac{Fab^2}{L^2} @ z=0$	$ \sigma_z  = \frac{ M_{Rz} }{I_x} u = \frac{ F b^2 u(zL+2za+aL)}{I_x L^3}$ $ \sigma_{max}  = \frac{ Fu ab^2}{L^2 I_x} @ z=0$ $ C_y  = \left  \frac{F}{\delta y_{z(F)}} \right  = \frac{3EI_x L^3}{b^2 a^3 (L-a)} @ z=a$
12	$\theta_A = 0$ $\theta_B = 0$ $\delta y_z = -\frac{qL^4}{24EI_x} \left[ \left( \frac{z}{L} \right)^2 - 2 \left( \frac{z}{L} \right)^3 + \left( \frac{z}{L} \right)^4 \right]$ $\delta y_{max} = -\frac{qL^4}{384EI_x} @ z = \frac{L}{2}$	$R_A = \frac{1}{2} qL$ $R_B = \frac{1}{2} qL$ $D_z = \frac{qL}{2} - qz$ $M_{Rz} = \frac{q(-6z^2 + 6Lz - L^2)}{12}$ $M_{Rmax} = -\frac{qL^2}{12} @ z=0$	$ \sigma_z  = \frac{ M_{Rz} }{I_x} u = \frac{ q (-6z^2 + 6Lz - L^2)}{12I_x}$ $ \sigma_{max}  = \frac{ q L^2 u}{12I_x} @ z=\frac{L}{2}$ $ C_y  = \left  \frac{q}{\delta y_{max}} \right  = \frac{384EI_x}{L^4} @ z = \frac{L}{2}$
	$\theta_A = 0$ $\theta_B = 0$ $\delta y_z = -\frac{Fl^2(3L-2z)}{12EI_x}$ $\delta y_{max} = -\frac{Fl^3}{12EI_x} @ z=L$	$R_A = F$ $R_B = 0$ $D_z = F$ $M_{Rz} = \frac{Fl-2Fz}{2}$ $M_{Rmax} = \frac{Fl}{2} \text{ resp. } -\frac{Fl}{2} @ z=0 \text{ resp. } L$	$ \sigma_z  = \frac{ M_{Rz} }{I_x} u = \frac{ u (FL-2Fz)}{2I_x}$ $ \sigma_{max}  = \frac{ F Lu}{2I_x} @ z=L$ $ C_y  = \left  \frac{F}{\delta y_{max}} \right  = \frac{12EI_x}{L^3} @ z=L$